

The crankshaft is vital, and a considerable apparatus part is having complex and superior geometry in the block of the engine, which transforms the reciprocating displacement of the IC piston to a rotating cyclic motion with a proper four-link system mechanism [4]. The paper needs to depict the better efficient fuel economical with lighter weight in high efficiency in output [4]. This study is based on a single cylinder with a four-stroke engine cycle [2]. Two different designed crankshafts from the same engines were considered in this proposed research. The finite element analysis (FEA) was used in four various static steps for each designed crankshaft. Stresses from the mechanical breakdown describe dynamic loading to the designed crankshafts. In the past study was done on the forged steel designed crankshaft to specify the weight optimization and the cost of manufacture [12].

II. CRANKSHAFT DESIGN FAILURE

The frequency and inter-area tie-line power should be near to the scheduled values for large scale power systems. The crankshaft is mainly designed as a component of an engine which converts the reciprocating motion of the piston into rotational motion [8]. The central rotating part of an engine is crankshaft, which is mainly designed from ductile iron [5]. The crankshaft consists of many features like crankpin journal, throw, bearing journals, counterweights, crank gear, and the power take-off (PTO). The rotating pivot point of the crankpin journal usually attaches the connecting rod to the crankshaft [19].

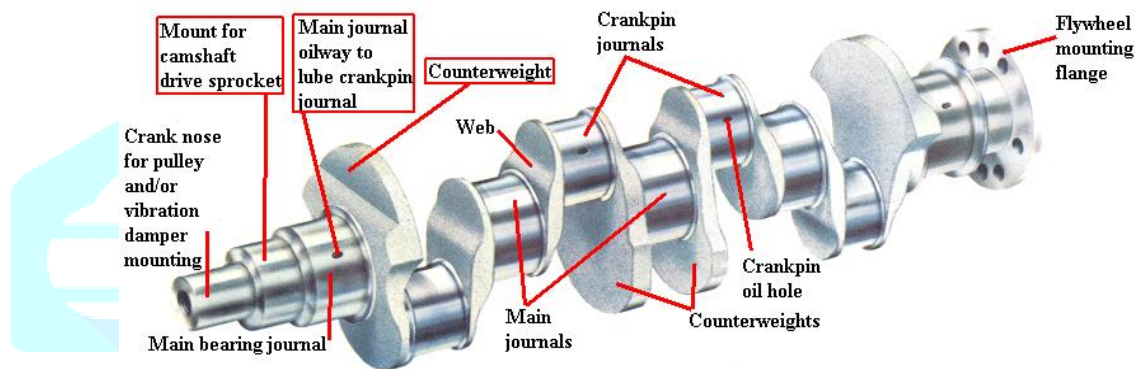


Figure 1.2: Crankshaft Design



Figure 1.8: Failure of Crankshaft

The three major sources like operating sources, mechanical sources and repairing sources are mainly responsible for the failure in the engine [35]. Fatigue is the most common failure in the crankshaft mainly occurs at fillet areas due to the bending loads triggered by the combustion effect. A twisting or torsional crack should be expected at pin surface, radius or oil whole surface of the crankshaft. Significant stress experiences at crankshaft fillet in its service life due to its geometry and engine system [37].

III. FINITE ELEMENT ANALYSIS

The finite element method is one of the most common numerical method used to find out the solution related to engineering and mathematical physics problems. The solution analysis mainly includes issues like stress analysis, electromagnetic and heat transfer. The finite element analysis can be used as a physical problem solver method in the field of loadings, complicated geometrics and material properties. In this numerical analysis method, the domain is segregated into smaller units or elements called finite elements. Hand calculation is the only method used in finite element analysis and structural development analysis.

Material Properties:

Table 5.1 Material properties of structural steel

Material Selected	Structural Steel
Density (kg/m ³)	7850
Young's Modulus (pa)	2e11
Yield strength (pa)	2.5e8
Poisson Ratio	0.3
Behaviour	Isotropic

Table 5.2 Material properties of Aluminum Alloy

Material Selected	Aluminium Alloy
Density (kg/m ³)	2770
Young's Modulus(pa)	7.11e10
Yield strength (pa)	2.8e8
Poisson Ratio	0.33
Behaviour	Isotropic

IV. STATIC STRUCTURAL ANALYSIS

A static analysis includes static equivalent loads like steady inertia loads and time-varying loads used for the calculation of loading effects by ignoring the inertia and damping effects of the loads. The high stress leads to crack formation, which is the leading cause for the failure of the crankshaft in a single cylinder engine. The stress value can be minimized by doing modification in the design with reduced weight, which leads to the lighter crankshaft. This stress minimization leads to the reduction of inertial and centrifugal forces.

Comparison of static structural FEA results for un-optimized crank by varying material:

Sl.No	Parameters	Existing Values	FEA results for Structural Steel	FEA results for Aluminium Alloy
1	Total deformation (m)	8.75e-3	5.24e-6	1.48e-5
2	Directional deformation (m)	4.78e-3	2.66e-7	8.45e-7
3	Von-misses strain (m/m)	2.56e-4	2.6e-4	7.54e-4
4	Von-misses stress(pa)	6.55e7	5.338e7	5.35e7
5	Shear stress (pa)	1.863e	1.707e7	1.68e7

6	Safety factor	4.7496	1.6148	1.5456
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V. ANSYS VALIDATION RESULTS

1.5.1 Static structural FEA results of varying Parameters for Structural Steel:

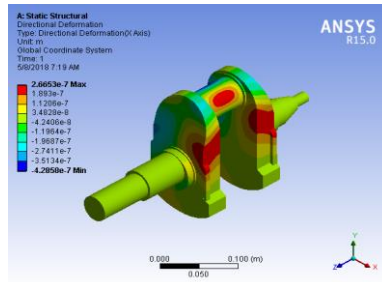
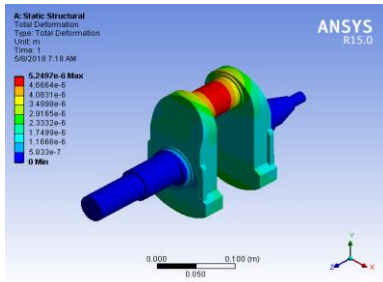


Figure 6.5: total deformation of crank Figure 6.6: Directional Deformation

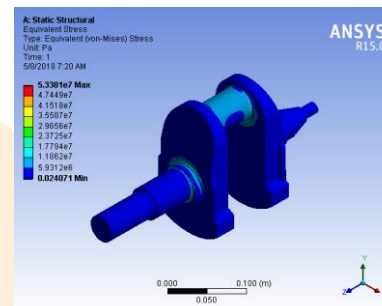
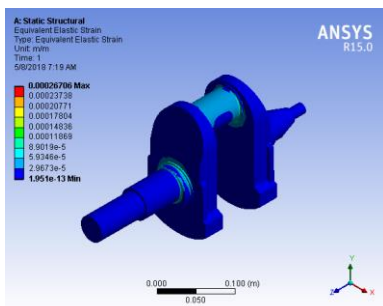


Figure 6.7: Equivalent Strain Figure 6.8: Von-Mises Stress

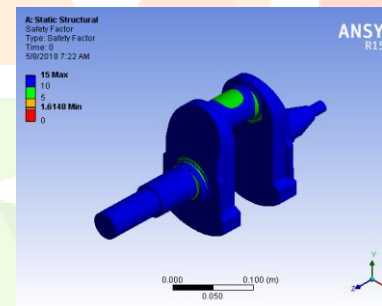
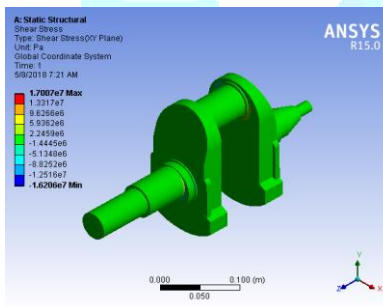


Figure 6.9: Shear Stress Figure 6.10: Safety Factor

1.5.2 Static structural FEA results of varying Parameters for Aluminium Alloy:

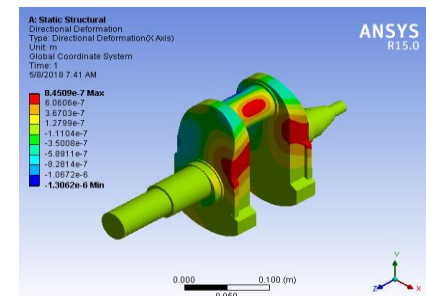
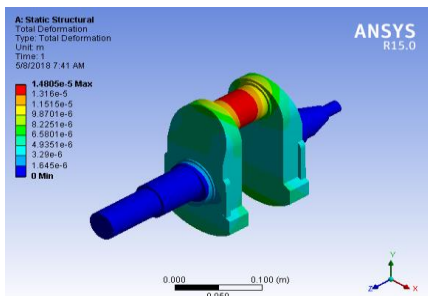


Figure 6.11: Total Deformation Figure 6.12: Directional Deformation

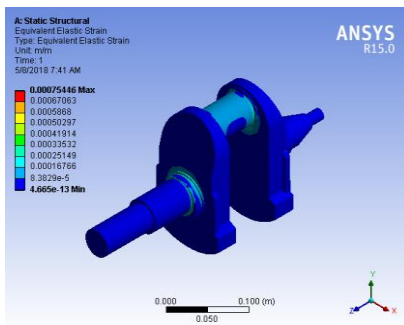


Figure 6.13: Elastic strain

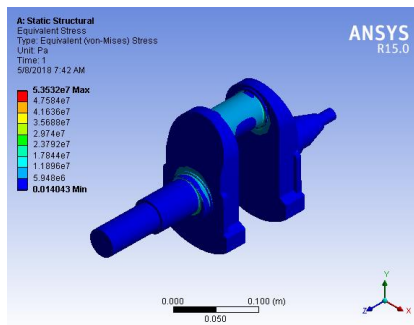


Figure 6.14: Von-Mises Stress

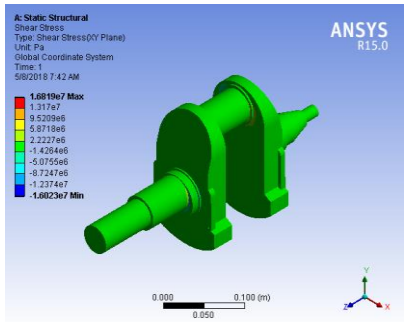


Figure 6.15: Shear Stress

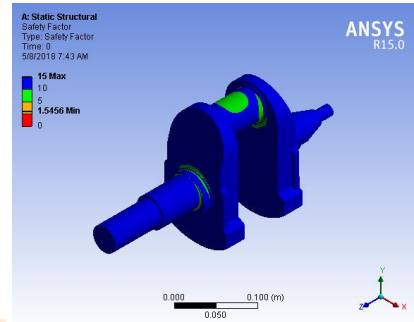


Figure 6.16: Factor of Safety

VI. CONCLUSION

The results obtained from the Finite element analysis (FEA) shows that the critical location of the crankshaft geometry discovered in the fillet region. The crankshaft failure may instigate at the crank pin and therefore in future scope, the fatigue phenomena regarded as the main reason for failure. The failure of the crankshaft can be studied by vibrational calculation. Thus from the above study, it has been concluded that the finite element method is the essential tool for designing and optimization of the crankshaft.

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