

# Design and Performance Enhancement of Induced Draft Counter Flow Wet Type Cooling Tower

<sup>1</sup>Dhruvin.Y.Patel<sup>2</sup>Prof. Harshal Oza<sup>3</sup>Prof.Jyotin Katashia<sup>1</sup>P.G Student<sup>2,3</sup>Assistant Professor,<sup>1,2,3</sup>Gandhinagar Institute of Technology,Gandhinagar, 382721

## Abstract:

Cooling towers are one of the biggest heat and mass transfer devices used to transfer process waste heat to the atmosphere. Cooling towers make use of evaporation whereby some of the water is evaporated into a moving air stream and subsequently discharged into the atmosphere. As a result, the remainder of the water is cooled down significantly.

The process parameters such as inlet Air Wet bulb Temperature, Flow rate of Water and fills porosity have more influence on Thermal performance of cooling tower. The Temperature of outlet water is maintained nearest to inlet air wet bulb temperature to obtain the best Thermal Performance of cooling tower.

## Nomenclature

$T_1$ -Inlet temperature of hot water circulating in cooling tower

$T_2$ -Outlet temperature of hot water circulating in cooling tower

$\rho$  -Mass density of water

H-Height of cooling tower

$T_{a1}$ -Inlet temperature of air

$C_{pw}$  -Specific heat of water

$H_{a1}$ -Enthalpy of air at inlet temperature

$W_1$ -Specific humidity of air at inlet temperature

$V_{s1}$ -Specific volume of air at inlet temperature

$H_{w1}$ -Enthalpy of water at inlet temperature

CTA-Cooling Tower Approach

HL-Heat Loss by Water

$M_{w1}$ -Mass of water circulated in cooling tower

$M_a$ -Mass of Air

K -Mass transfer co-efficient

$\eta$ - Effectiveness of Cooling Tower

WL -Windage Losses

WBT-Wet Bulb Temperature

$\phi$  -Relative humidity

R-Constant

D-Diameter of cooling tower

$T_{a2}$  - Outlet temperature of air

K-Thermal conductivity of steel

$H_{a2}$  - Enthalpy of air at outlet temperature

$W_2$ -Specific humidity of air at outlet temperature

$V_{s2}$ -Specific volume of air at outlet temperature

$H_{w2}$  -Enthalpy of water at outlet temperature

CTR -Cooling Tower Range

V -Volume of Air

HG-Heat Gain

$M_{mak}$  - Mass of Makeup Water

$d_i$ -Inside diameter of pipe

DL-Drift Losses

EL-Evaporation Losses

## 1. INTRODUCTION

### 1.1 Cooling Tower

Cooling towers are heat exchangers which are used to dissipate large heat loads to the atmosphere. It is equipment used to reduce the temperature of a water stream by extracting heat from water and emitting it to the atmosphere. They are used in a variety such as power generation and refrigeration. Cooling towers are designed for industrial plants for various purposes and sizes to provide cool water. Typically, a condenser of a power plant and or of heating ventilation, and air conditioning (HVAC) system is cooled by water.

## 1.2 Induced Draft

An induced draft mechanical draft tower is a draw-through arrangement, where a fan located at the discharge end pulls air through the tower. The fan induces hot moist air out the discharge. This produces low entering and high exiting air velocities, reducing the possibility of recirculation in which discharged air flows back into the air intake.

## 1.3 Counter Flow Cooling Tower

In a counter-flow induced draft cooling tower, air travels vertically across the fill sheet, opposite to the downward motion of the water. Air enters an open area beneath the fill media and is then drawn up vertically. The water is sprayed through pressurized nozzles and flows downward through the fill, opposite to the air flow.

## 1.4 Cooling Tower Performance [4]

The important parameters, from the point of determining the performance of cooling towers, are:

1. "Range" is the difference between the cooling tower water inlet and outlet temperature.
2. "Approach" is the difference between the cooling tower outlet cold water temperature and ambient wet bulb temperature.
3. Cooling tower effectiveness (in percentage) is the ratio of range, to the ideal range, i.e., difference between cooling water inlet temperature and ambient wet bulb temperature, or in other words it is =  $\text{Range} / (\text{Range} + \text{Approach})$ .
4. Cooling capacity is the heat rejected in kCal/hr or TR, given as product of mass flow rate of water, specific heat and Temperature difference.
5. Evaporation loss is the water quantity evaporated for cooling duty. \*Evaporation Loss (m<sup>3</sup>/hr) =  $0.00085 \times 1.8 \times \text{Circulation rate (m}^3/\text{hr)} \times (T_1 - T_2)$  T<sub>1</sub>-T<sub>2</sub> = Temperature Difference between inlet and outlet water.

## 2. METHODOLOGY

### 2.1 Design Calculation [1]

#### 2.1.1 Cooling Tower Approach (CTA)

CTA: Cooling water outlet temperature - WBT

$$\text{CTA} : 33 - 30$$

$$\text{CTA} : 3^{\circ}\text{C}$$

#### 2.1.2 Cooling Tower Range (CTR)

$$\text{CTR: } T_1 - T_2$$

$$\text{CTR: } 38 - 33$$

$$\text{CTR: } 5^{\circ}\text{C}$$

Now, Mass of water circulated in cooling tower

$M_{w1} = \text{Volume of circulating water} \times \text{Mass density of water}$

$$M_{w1} = 10 \times 1000$$

$$M_{w1} = 10000 \text{ Kg / hr}$$

#### 2.1.3 Heat Loss by Water (HL)

$$\text{HL: } M_{w1} \times C_{pw} \times (T_1 - T_2)$$

$$\text{HL: } 10000 \times 4.186 \times (38 - 33)$$

$$\text{HL: } 209300 \text{ KJ / hr}$$

### 2.1.4 Volume of Air Required (V)

$$V = \frac{(HL \times V_{s1})}{[(H_{a2} - H_{a1}) - (W_2 - W_1) \times C_{pw} \times T_2]}$$

$$V = \frac{(209300 \times 0.908)}{[(143.83 - 99.73) - (0.0391 - 0.0252) \times 4.186 \times 33]}$$

$$V = \frac{190044.4}{42.179}$$

$$V = 4505.56 \text{ m}^3 / \text{hr}$$

### 2.1.5 Heat Gain by Air (HG)

$$HG = \frac{V \times [(H_{a2} - H_{a1}) - (W_2 - W_1) \times C_{pw} \times T_2]}{V_{s1}}$$

$$HG = \frac{4505.56 \times [(143.83 - 99.73) - (0.0391 - 0.0252) \times 4.186 \times 33]}{0.908}$$

$$HG = \frac{190044.07}{0.908}$$

$$HG = 209300 \text{ KJ} / \text{hr}$$

### 2.1.6 Mass of Air Required (M<sub>a</sub>)

$$M_a = \frac{\text{Volume of air required}}{\text{Specific volume of air at inlet temperature}}$$

$$M_a = \frac{V}{V_{s1}}$$

$$M_a = \frac{4505.56}{0.908}$$

$$M_a = 4962.07 \text{ Kg} / \text{hr}$$

### 2.1.7 Effectiveness of Cooling Tower

$$\eta = \frac{(T_1 - T_2)}{(T_1 - \text{WBT})}$$

$$\eta = \frac{(38 - 33)}{(38 - 30)}$$

$$\eta = 62.5\%$$

## 2.2 Different Types of Losses Generated In Cooling Tower [3]

### 2.2.1 Drift Losses (DL)

Drift losses are generally taken as 0.10 % (Perrys chemical engineering hand book) of circulating water.)

$$DL = 0.10 \times m_{w1} / 100$$

$$DL = 0.10 \times 10000 / 100$$

$$DL = 10.0 \text{ Kg / hr}$$

### 2.2.3 Wind age Losses (WL)

Wind age losses are generally taken as 0.005(Perrys chemical engineering hand book) of circulating water.)

$$WL = 0.005 \times m_{w1}$$

$$WL = 0.005 \times 10000$$

$$WL = 50 \text{ Kg / hr}$$

### 2.2.4 Evaporation Losses (EL)

Evaporation losses are generally taken as 0.00085(Perrys chemical engineering hand book) of circulating water.)

$$EL = 0.00085 \times m_{w1} \times (T_1 - T_2)$$

$$EL = 0.00085 \times 10000 \times (38 - 33)$$

$$EL = 42.5 \text{ Kg / hr}$$

Water balance equation for cooling tower is

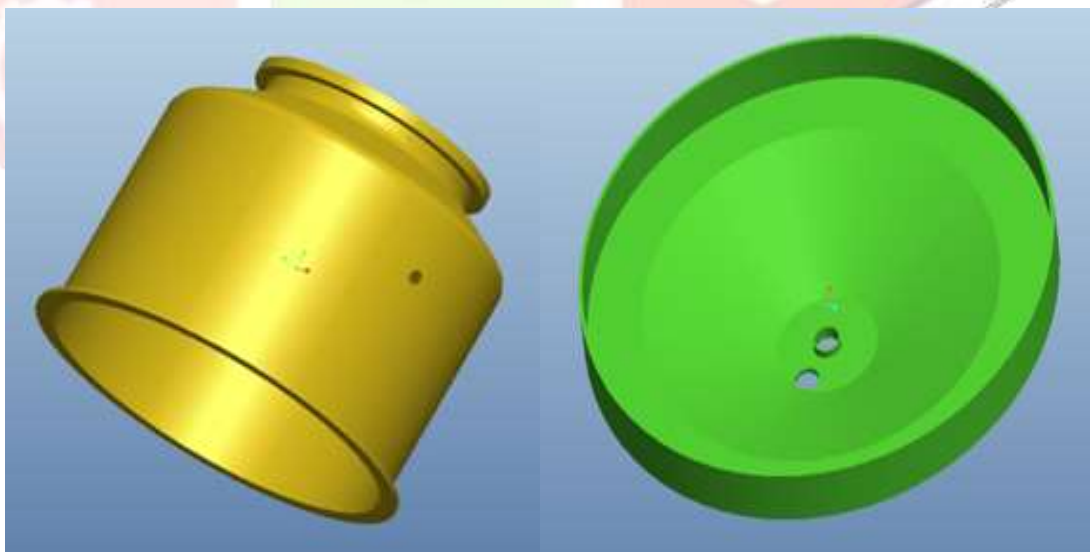
$$M = DL + WL + EL$$

$$M = 10 + 50 + 42.5$$

$$M = 102.5 \text{ Kg / hr}$$

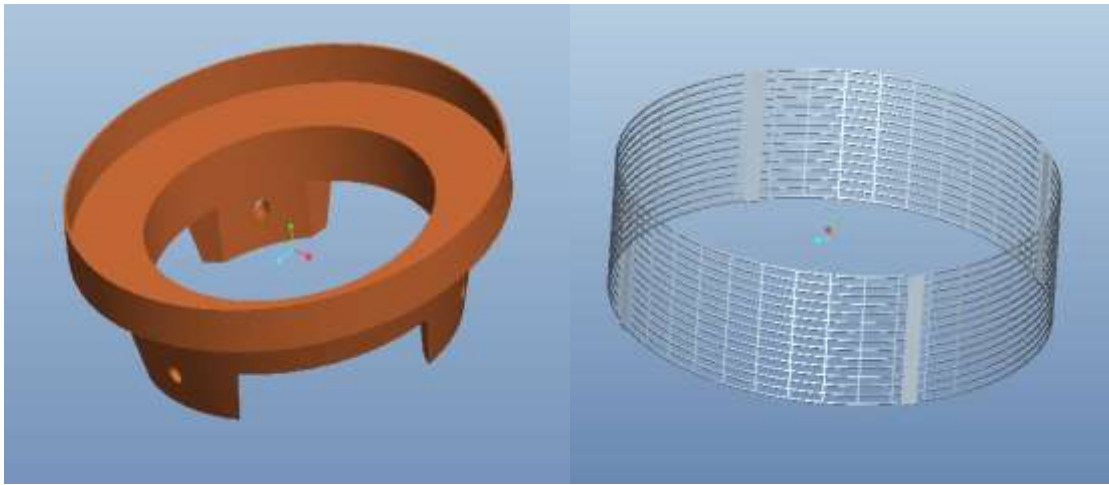
## 3. DESIGN OF COOLING TOWER

### 3.1 Different Design parts of cooling tower



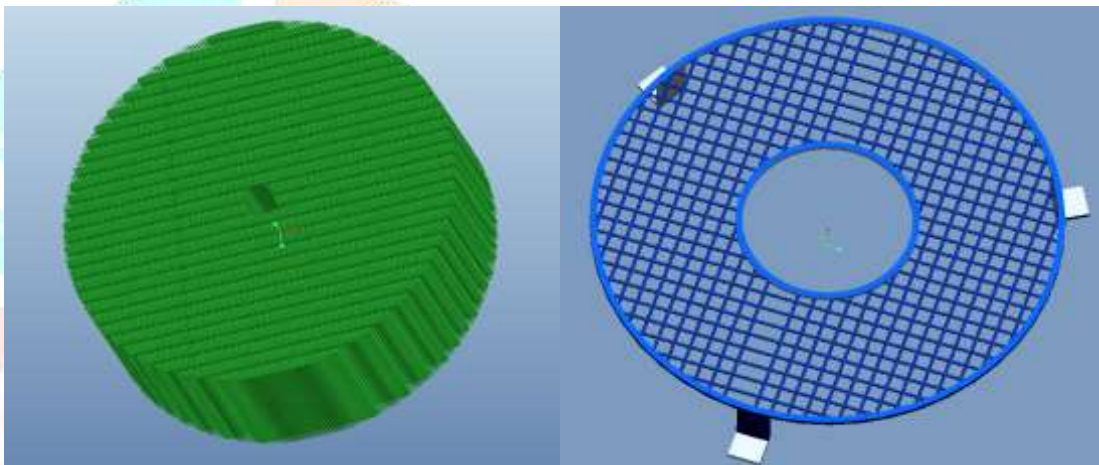
**Main cover**

**Basin**



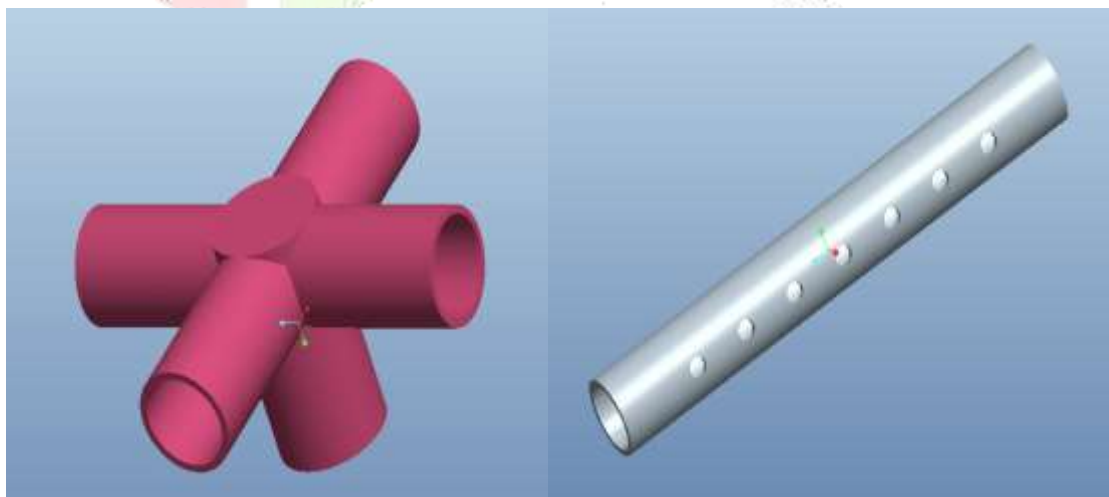
**Base Stand**

**Air intake**



**Fills**

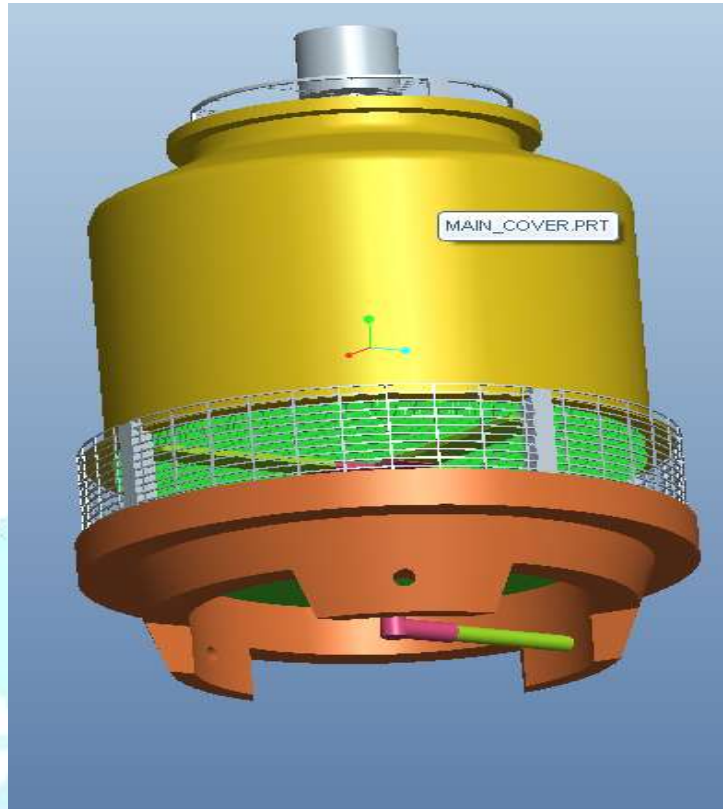
**Drift Eliminator**



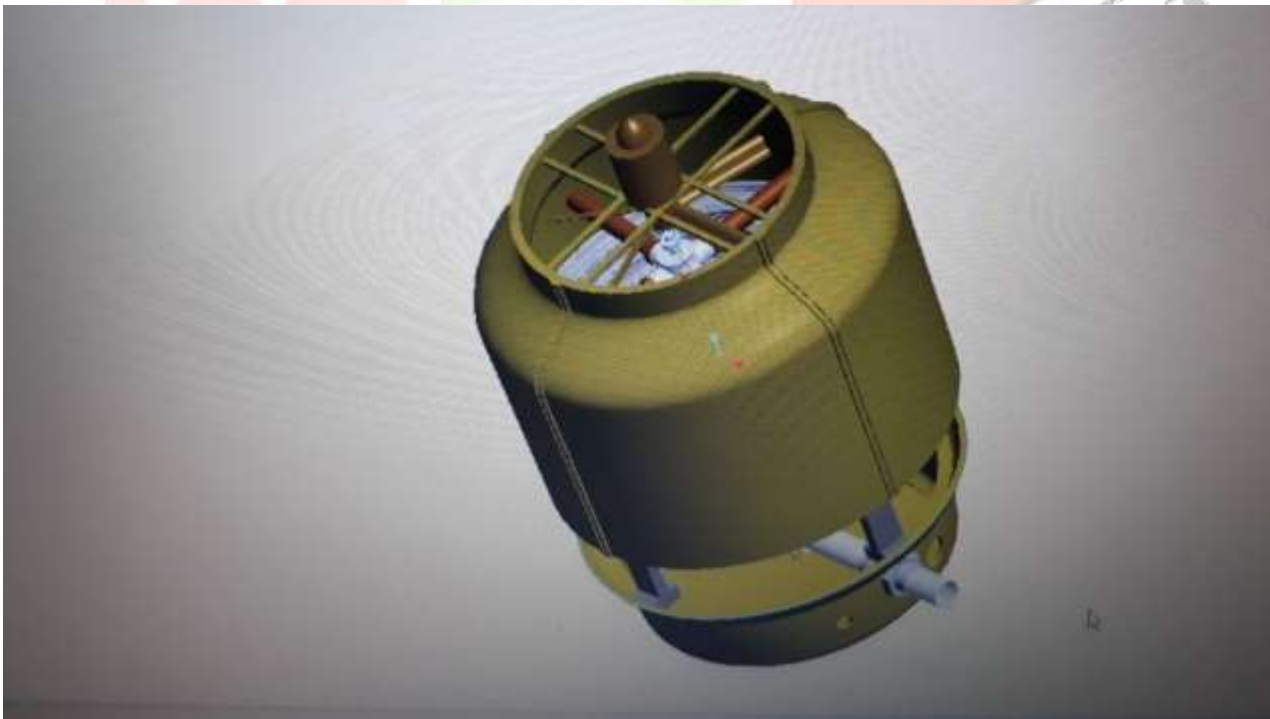
**Upper pipe**

**Sprinkler**

### 3.2 Assembly parts join of the Cooling tower



### 3.3 3D modelling of cooling tower



#### 4. REFERENCE

- [1] W.H. Walker, W.K. Lewis, W.H. McAdams, E.R. Gilliland, Principles of Chemical Engineering, third ed., McGraw-Hill Inc., New York, (1923).
- [2] Kloppers, J.C., and Kroger, D.G. A critical investigation into the heat and mass transfer analysis of counter flow wet cooling towers, International Journal of Heat and Mass Transfer, 48 (2005), 765–777.
- [3] Ronak Shah & Trupti Rathod Thermal design of cooling tower (International Journal of Advanced Engineering Research and Studies)
- [4] Ramakrishnan Ramkumar and Arumugam Ragupathy. “Thermal performance of forced draft counter flow wet cooling tower with expanded wire mesh packing” (International Journal on Technical and Physical Problems of Engineering) (IJTPE)
- [5] Jin-Kuk Kim, Robin Smith. Cooling water system design, Chemical Engineering Science, 56 (2001), 3641–3658.
- [6] G.Gan, S.B. Riffat, L.Shao, P.Doherty. Application of CFD to closed-wet cooling towers, Applied Thermal Engineering 21 (2001), 79-92. 7] Ali Reza Keyvani Boroujeni. Evaluation of FRP (fiberglass reinforced plastic) and RC (rapid cooling) cooling tower, Journal of Mechanical Engineering Research, 3 (2011), 152-156.
- [8] Velimir Stefanovic. Experimental Study on Heat and Mass Transfer in Cooling Towers, Mechanical Engineering, 7 (2000), 849 – 861.
- [9] Bhupesh Kumar. Experimental Study of the Performance of Cooling Tower (International Journal of Science and Research (IJSR))
- [10] Panjeshahi, Mohammad Hassan. A Comprehensive Approach to an Optimum Design and Simulation Model of a Mechanical Draft Wet Cooling Tower, Chemical journal, 29 (2010), 1-12.